

Renewable Energy: Geothermal Solution

SCHOOL OF ENGINEERING
MECHANICAL ENGINEERING

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Problem 1

- (a) The mass of the air to be heated is:

$$m_{\text{air}} = V \cdot \rho_{\text{air}} = 8000 \text{ m}^3 \cdot 1.29 \text{ kg/m}^3 = 10320 \text{ kg}$$

The required energy change:

$$\Delta \tilde{E} = m_{\text{air}} \cdot c_{\text{air}} \cdot \frac{\Delta T}{\Delta t} = 10320 \text{ kg} \cdot 1000 \frac{\text{J}}{\text{kg} \cdot \text{K}} \cdot 7.3 \cdot 10^{-3} \text{ K/s} = 75336 \text{ J/s}$$

Installed heating capacity:

$$P_n = 75.3 \text{ kW}$$

- (b) COP of heat pump = 4.2

E = Electrical energy (for pump)

Q_n = Useful heat

Q_u = Ambient heat

$$Q_n = Q_u + E$$

Coefficient of power for heat pump is defined as:

$$\text{COP} = \frac{Q_n}{E} = \frac{P_n}{P_{\text{el}}}$$

This implies electrical power:

$$P_{\text{el}} = \frac{75.3 \text{ kW}}{4.2} = 17.9 \text{ kW}$$

Heating power of the probe corresponds to Q_n , hence:

$$P_n = P_u + P_{\text{el}} \implies P_u = 75.3 \text{ kW} - 17.9 \text{ kW} = 57.4 \text{ kW}$$

- (c) l is the length of the probe. Using P_u from part (b):

$$l \cdot 52 \frac{\text{W}}{\text{m}} = 57.4 \text{ kW} \implies l = 1104 \text{ m}$$

We also have:

$$\frac{Q_u}{l} = \frac{Q_n - E}{l} = \frac{Q_n - \frac{Q_n}{\text{COP}}}{l}$$

Insertion of annual heating demand $Q_n = 135000 \text{ kWh}$ gives:

$$\frac{Q_u}{l} = 93.5 \frac{\text{kWh}}{\text{m}} < 110 \frac{\text{kWh}}{\text{m}}$$

- (d)

$$E = \frac{Q_n}{\text{COP}} = 32143 \text{ kWh}$$

The electricity cost:

$$32143 \text{ kWh} \cdot 0.13 \text{ CHF/kWh} = 4178.59 \text{ CHF}$$

(e) Oil heating: Annual heating demand in MJ:

$$135000 \text{ kWh} = 486000 \text{ MJ}$$

Volume of oil required:

$$V_{\text{oil}} = \frac{486000 \text{ MJ}}{42.6 \text{ MJ/kg} \cdot 0.86 \text{ kg/L}} = 13266 \text{ litres}$$

The price of the oil:

$$13266 \text{ litres} \cdot 0.86 \text{ CHF/litre} = 11408 \text{ CHF}$$

Problem 2

(a) Available heat in the geothermal source:

$$\dot{Q} = \dot{m} c_p (T_{\text{in}} - T_{\text{out}}) = 50 \cdot 4180 \cdot (190 - 85) = 21.945 \text{ MW}$$

Exergy available in the geothermal source:

$$T_{\text{logmean}} = \frac{T_{\text{h,in}} - T_{\text{h,out}}}{\ln \frac{T_{\text{h,in}}}{T_{\text{h,out}}}} = \frac{463 - 358}{\ln \frac{463}{358}} = \frac{105}{0.2572} = 408.25 \text{ K}$$

$$Ex_{\text{source}} = \dot{Q} \cdot \left(1 - \frac{T_a}{T_{\text{logmean}}}\right) = 21.945 \cdot \left(1 - \frac{287}{408.25}\right) = 21.945 \cdot (1 - 0.703) = 6.52 \text{ MW}$$

Exergy for district heating:

$$T_{\text{logmean}} = \frac{T_{\text{h,in}} - T_{\text{h,out}}}{\ln \frac{T_{\text{h,in}}}{T_{\text{h,out}}}} = \frac{333 - 313}{\ln \frac{333}{313}} = \frac{20}{0.06194} = 322.9 \text{ K}$$

$$Ex_{\text{heating}} = \dot{Q} \cdot \left(1 - \frac{T_a}{T_{\text{logmean}}}\right) = 12 \cdot \left(1 - \frac{287}{322.9}\right) = 12 \cdot (1 - 0.889) = 1.334 \text{ MW}$$

Energy Efficiency:

- Summer: $\frac{3.2 \text{ MW}}{21.945 \text{ MW}} = 14.6\%$
- Winter: $\frac{2.4 \text{ MW} + 12 \text{ MW}_{\text{th}}}{21.945 \text{ MW}} = 65.6\%$

Exergy Efficiency:

- Summer: $\epsilon = \frac{Ex_{\text{electrical}} + Ex_{\text{heating}}}{Ex_{\text{source}}} = \frac{3.2 + 0}{6.52} = 49.1\%$
- Winter: $\epsilon = \frac{Ex_{\text{electrical}} + Ex_{\text{heating}}}{Ex_{\text{source}}} = \frac{2.4 + 1.334}{6.52} = 57.3\%$

(b) The marginal electrical efficiency in winter is the electrical production from the residual heat ($21.945 \text{ MW} - 12 \text{ MW}_{\text{DH}} = 9.945 \text{ MW}$), thus:

$$\frac{2.4}{9.945} = 24.1\%$$